#### **DEVELOPMENT AND EVALUATION OF PINEAPPLE**

## (Ananas comosus [L.] Merr.) POMACE BASED

# EXTRUDER

BY

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# CERTIFICATION

I certify that this research work was carried out by Mr. O. B. Oduntan in the Department of Agricultural and Environmental Engineering of University of Ibadan.

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#### ABSTRACT

In large scale juice processing, about 65% of the pineapple is extracted as juice, while the remaining 35% is the by-product called pineapple pomace. Pomace generated from juice processing constitutes a disposal problem and environmental pollution and there is no processing equipment designed in addressing this problem through the utilisation of this byproduct. This study was designed to develop a mash extruder for pineapple pomace based flour.

Pineapple pomace was analysed for its proximate composition using standard methods. Pomace flour was produced by drying and grinding fresh pomace from a juice processing plant. The physical and mechanical properties of the pineapple pomace flour using cassava flour as the binder were carried out by standard methods. An experimental laboratory press was used to evaluate the extrusion point pressure at different pineapple pomace/cassava flour ratio mash (5:1, 6:1 and 7:1), the moisture content of the mash (50, 55 and 60%), die size (4, 6 and 8 mm) and temperatures (60, 80, 100 and 120°C). Response Surface Method was employed to optimise the experimental data with extrusion point pressure as the response variable while temperature, moisture content, die size and pomace ratio were the independent variables. The results obtained were used to design and test a single screw pineapple pomace extruder. Operational power, compression ratio, throughput and efficiency of the extruder were determined using standard methods. Data were analysed using ANOVA.

Pineapple pomace contained 12.4% moisture content, 4.8% ash, 1.4% fat, 9.2% crude protein, 6.0% crude fibre and 66.2% carbohydrate. The static angle of repose and coefficient of friction at various combinations of pomace and cassava flours increased linearly for the entire surface with moisture content and varied with structural surface in the moisture range of 12.4 to 26.3% (d.b). The minimum value of coefficient of friction for stainless iron steel was 0.44 for pineapple pomace based flour. The extrusion point pressure was 7.51  $\pm$  0.62MPa, temperature, die sizes and pomace ratio significantly (p < 0.05) influenced extrusion point pressure. Optimum conditions for the extrusion point pressure were 100°C; 4.0 mm die size, 55.0% moisture content and 6:1 pomace ratio at maximum desirability of 1:00. The power required to operate the extruder was 4.0 kW at a compression ratio of 3:1. Machine throughput was 26.1 kg/h with the extruding efficiency of 87.9%.

An efficient 4.0 kW pineapple pomace based extruder has been developed, which can be used to process and conserve pineapple pomace.

Keywords: Pineapple pomace-flour, Extruder-development, Point-pressure.

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# DEDICATION

# To My God

Who Make My Way Prosperous, Wisely and Successful

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# LIST OF ABBREVIATIONS

Symbol/ Abbreviation	Meaning	Unit
$\mathbf{V}_{f}$	volume of flask containing the digest x 100	m <sup>3</sup>
W <sub>sd</sub>	weight of sample digested	mg
V <sub>ds</sub>	vol. of digest for steam distillation	m <sup>3</sup>
W <sub>as</sub>	weight of ash	g
W <sub>o</sub>	original weight of sample	g
MC <sub>db</sub>	moisture content (dry basis)	%
MC <sub>wb</sub>	moisture content (wet basis)	%
$m_d$	mass of dry matter	kg
$m_w$	mass of water to be added.	kg
ds	diameter of the screw shaft	mm
Т	Torque transmitted by the shaft of extruder	Nm
δο	yield stress for mild steel.	Nmm <sup>2</sup>
Un	screw depth at the discharge end	mm
a	screw depth at the feed end	mm
n	number of screw turns	unit less
We	load that can be lifted by the screw	kN
Т	Torque transmitted by the screw shaft	Nm
D <sub>m</sub>	mean thread diameter	mm
μ	coefficient of friction	unit less
$\theta_n$	thread (lift) angle	$0^{\mathrm{o}}$
α	tapering angle	$0^{\mathrm{o}}$

P <sub>r</sub>	pressure developed by the screw threads	N/mm <sup>2</sup>
A <sub>p</sub>	pressing area	$\mathrm{mm}^2$
h	screw depth at the maximum pressure	mm
P <sub>b</sub>	pressure to be withstood by the barrel	N/mm <sup>2</sup>
t	thickness of the barrel	mm
$\delta_a$	allowable stress	N/mm <sup>2</sup>
$\delta_{o}$	yield stress of mild steel	N/mm <sup>2</sup>
D <sub>i</sub>	inside diameter of the barrel	mm
Qt	theoretical volumetric capacity	$m^{3/s}$
D <sub>sf</sub>	diameter of the screw flight	m
d <sub>ss</sub>	base diameter of the screw shaft	m
Ps	screw pitch	m
N <sub>s</sub>	rotational speed	rw/s
Pe	power required to drive the expeller	kW
Qt	volumetric capacity of the worm shaft	$m^{3/s}$
ls	length of screw shaft	mm
g	acceleration due to gravity	m/s <sup>2</sup>
F	material factor	unit less